

TAPTITE 2000®
Thread Rolling Fasteners

Unique Design Increases Performance

TAPTITE 2000® fasteners are designed to provide the benefits of previous TAPTITE® fastener products with an innovative new thread design – the Radius Profile™ Thread. The proven TRILOBULAR® principle is maintained while incorporating this beneficial thread design. The result is a new generation of TAPTITE® fasteners, providing excellent mechanical, assembly, and ergonomic characteristics surpassed by no other technology. TAPTITE 2000® fasteners afford end-users with enhanced opportunities to reduce the overall cost of assembly.



TRILOBULAR® Configuration

- Reduces friction
- Increases prevailing torque
- Resists loosening caused by vibration
- Lower end load requirements

Radius Profile™ Thread

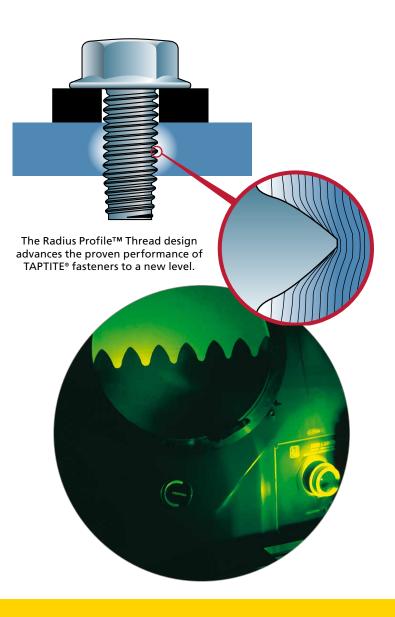
- Lowers thread forming torque without sacrificing performance
- Higher, more uniform drive-to-fail ratio
- Increased drive-to-strip ratio
- Resist internal thread stripping
- Excellent axial alignment

Roll Forms Own Work-Hardened Mating Threads

 Results in higher strength internal threads due to the cold flow/work hardening that occurs during the forming of the nut thread

Available with TORX PLUS® Drive System

- Significantly extends tool life
- Ideal drive system for maximum torque transfer



Built On Proven Performance

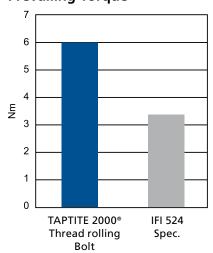


TAPTITE® fasteners have been used in a variety of industries since its inception over forty years ago. Built upon those years of experience with the proven performance features of TAPTITE®, TAPTITE II®, and Duo-TAPTITE® fasteners, the TAPTITE 2000® design combines all these previous benefits with an even lower thread-forming torque. The result is optimal joint performance in a wide range of applications.

Application Comparison	TAPTITE II® Fasteners	Duo-TAPTITE® Fasteners	TAPTITE 2000® Fasteners
Lowest thread-forming torque			х
Positioning applications	x		x
Structural applications		x	x
High torque-tension applications		x	х
High strip-out requirement		x	х
Automation capability		x	x
Nut member greater than 1.5 x screw dia.	х		х
Nut member less than 0.3 x screw dia.		x	x
Adaptability to different point styles	x		x
High axial alignment required		x	X

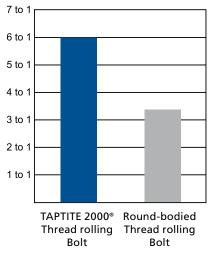
TRILOBULAR® TAPTITE 2000® Fasteners vs. Other Thread Rolling Fasteners

Prevailing Torque



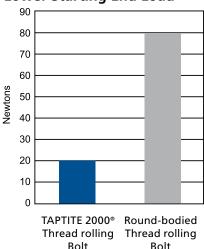
The TRILOBULAR® shape of TAPTITE 2000® provides high prevailing torque. Graph shows comparison of a TAPTITE 2000® fastener with IFI-524 locking screw specification.

Higher Drive-to-Fail Ratio



The higher, more uniform drive-to-fail ratio of TAPTITE 2000® bolts provide a built-in safety factor against over-torquing.

Lower Starting End Load



TAPTITE 2000® fasteners require a low axial end load to initiate thread forming.

Design Specifications







Design Specifications for TAPTITE 2000® Fasteners

- Diameters: M1.6 M16 (#2 5/8")
- **Head Styles:** Pan, hex washer, flat, oval, round washer, button head, fillister and specials
- Drive Systems: Any drive system, including the TORX PLUS® Drive System
- Point Styles: Standard TAPTITE® point, "CA" point, or "SP"™ point
- **Specials:** Shoulder screws, double-end studs, collar studs, sems; others may be available
- Materials: Low carbon steel, medium carbon steel, alloy steel, stainless steel; others may be available
- **Heat Treat:** Camcore® fasteners, case hardening or through hardening (see details below)

Please contact a STANLEY Engineered Fastening applications engineer for information on specials and assistance in selecting the optimal fastener for your application.

TAPTITE 2000® Heat Treatment

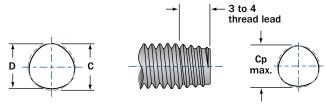
- Camcore® Fasteners: When tapping into high strength steel or in structural applications, a Camcore® fastener is recommended. Camcore® fasteners are made from high strength alloy steel which is through hardened to HRC 33-39 and tempered. The fastener lead threads are induction hardened to a minimum of HRC 45 by a secondary induction treatment. When combined with non-electroplated finishes, resistance to stress corrosion cracking is improved.
- Case Hardening: Case hardening is the standard heat treatment for all TAPTITE 2000® screws in sizes M5 (#12) and smaller. This process raises the surface hardness to a level higher than the core hardness.
- Through Hardening: Through hardening, which increases toughness and ductility, is the preferred treatment for TAPTITE 2000® fasteners used in aluminum applications.

TORX PLUS® Drive System

TAPTITE 2000® fastener performance is improved with the TORX PLUS® Drive System. Its 0° drive angle provides a more efficient transfer of torque, improving driveability. Its reduced drive tool tolerance and broad engagement area increases tool bit fatigue life, so drive tools last longer.

Many styles are available, including internal, external, low-profile external, tamper-resistant, and dual drives. The AUTOSERT® feature, which allows for high RPM engagement in internal drives, can be included.

Dimensional Data



For M5 (#12) and smaller TAPTITE 2000® fasteners have a special point design featuring a long lead (3-4 threads) for low thread-forming torque.

Section A-A Moderate Lobulation A + 4 to 5 thread lead Cp max. Section B-B Generous Lobulation

Larger sizes, M6 (#12) and larger, have stabilizing threads to aid alignment and ease starting.

Metric Data

	Screw Body	Dimensions	Point
	С	D	Ср
Screw Size	nom.	nom.	max.
M1.6 x 0.35	1.60	1.56	1.40
M2.0 x 0.40	2.00	1.96	1.77
M2.5 x 0.45	2.50	2.45	2.25
M3 x 0.5	3.00	2.95	2.71
M3.5 x 0.6	3.50	3.44	3.17
M4 x 0.7	4.00	3.93	3.60
M5 x 0.8	5.00	4.92	4.55
M6 x 1.0	6.00	5.90	5.38
M8 x 1.25	8.00	7.87	7.23
M10 x 1.5	10.00	9.85	9.08
M12 x 1.75	12.00	11.82	10.92
M14 x 2.0	14.00	13.80	12.77
M16 x 2.0	16.00	15.80	14.76

Length Tolerance

Metric per ANSI B18.6.7M

Nominal Screw Length	Tolerance on Length
to 3mm inclusive	±0.2mm
over 3 to 10mm inclusive	±0.3mm
over 10 to 16mm inclusive	±0.4mm
over 16 to 50mm inclusive	±0.5mm
over 50mm	±1.0mm

Inch Data

	Screw Body	Point	
	С	D	Ср
Screw Size	nom.	nom.	max.
2-56	.086	.084	.077
3-48	.099	.097	.088
4-40	.112	.110	.098
5-40	.125	.123	.111
6-32	.138	.135	.121
8-32	.164	.161	.147
10-24	.190	.186	.167
10-32	.190	.187	.174
12-24	.216	.212	.193
1/4-20	.250	.245	.220
5/16-18	.313	.307	.279
3/8-16	.375	.369	.337
7/16-14	.438	.431	.394
7/16-20	.438	.433	.407
1/2-13	.500	.492	.453
9/16-12	.563	.555	.511
5/8-11	.625	.616	.569

Length Tolerance

Inch per ANSI B18.6.3

	Nominal S	crew Size
	#4 – #12	1/4" – 1/2"
Nominal Screw Length	Tolerance	on Length
To 1/2" inclusive	+0,020"	+0,030"
Over 1/2" to 1" inclusive	+0,030"	+0,030"
Over 1" to 2" inclusive	+0,060"	+0,060"
Over 2"	+0,090"	+0,090"

Recommended Pilot Hole Sizes in Steel Nut Members – Metric Sizes (mm)

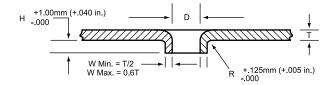
Application Duty Class	Light Medium-Light 0.3 Dia. of Material 0.5 Dia. of Material					Medium-Heavy 0.75 Dia.s of Material			trengt of Mat		Extended 1.25 Dia. of Material				
Percent. of Thread		90%		80%			70%			65%			60%		
Nominal Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size
M2.5 x 0.45	0.5 - 0.9	2.24	2.25	0.9 - 1.5	2.27	#43 2.26	1.5 - 2.1	2.3	2.3	2.1 - 2.7	2.31	2.3	2.7 - 3.5	2.32	2.3
M3 x 0.5	0.5 - 1.1	2.71	#36 2.71	1.1 - 1.7	2.74	2.75	1.7 - 2.7	2.77	7/64	2.7 - 3.3	2.79	7/64	3.3 - 4.0	2.8	2.8
M3.5 x 0.6	0.6 - 1.4	3.15	1/8 3.18	1.4 - 2.0	3.19	3.2	2.0 - 2.9	3.23	3.25	2.9 - 3.8	3.25	3.25	3.8 - 4.5	3.27	#30
M4 x 0.7	0.8 - 1.4	3.59	3.6	1.4 - 2.4	3.64	#27 3.66	2.4 - 3.3	3.68	3.7	3.3 - 4.4	3.7	3.7	4.4 - 5.5	3.73	#26
M4.5 x 0.75	0.9 - 1.7	4.06	#21 4.04	1.7 - 2.7	4.11	4.1	2.7 - 3.9	4.16	4.2	3.9 - 4.9	4.18	4.2	4.9 - 6.4	4.21	4.2
M5 x 0.8	1.0 - 2.1	4.53	4.5	2.1 - 2.9	4.58	4.57	2.9 - 4.4	4.64	#14 4.62	4.4 - 5.9	4.66	4.65	5.9 - 7.1	4.69	4.7
M6 x 1.0	1.2 - 2.4	5.42	#3 5.41	2.4 - 3.6	5.48	5.5	3.6 - 4.9	5.55	7/32 5.56	4.9 - 6.9	5.58	5.6	6.9 - 8.1	5.61	5.6
M7 x 1.0	1.4 - 2.4	6.42	6.4	2.4 - 4.4	6.48	6.5	4.4 - 6.5	6.55	F 6.53	6.5 - 7.7	6.58	6.6	7.7 - 9.5	6.61	6.6
M8 x 1.25	1.6 - 3.1	7.27	7.25	3.1 - 4.9	7.35	L 7.37	4.6 - 6.9	7.43	7.4	6.9 - 8.9	7.47	M 7.49	8.9 - 10.9	7.51	7.5
M10 x 1.5	1.9 - 3.9	9.12	23/64 9.1	3.9 - 5.9	9.22	9.25	5.9 - 8.3	9.32	9.3	8.3 - 10.9	9.37	U 9.35	10.9 - 12.9	9.41	9.4
M12 x 1.75	2.4 - 4.9	10.98	11.0	4.9 - 7.4	11.09	7/16 11.11	7.4 - 10.5	11.2	7/16 11.11	10.5 - 14.5	11.26	11.3	14.5 - 17.0	11.31	11.3

Recommended Pilot Hole Sizes in Steel Nut Members – Inch Sizes (in.)

Application		Light		Med	ium-Li	ght	Medi	um-He	avy	Full	Streng	th	Extended		
Duty Class	0.3 Dia	. of Ma	terial	0.5 Dia	. of Ma	terial	0.75 Dia.s of Material			1.0 Dia	. of Ma	terial	1.25 Dia. of Material		
Percent. of Thread	90% 80%				70%				65%		60%				
Nominal Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size	Material Thckns.	Pilot Hole	Drill Size
2-56	.017034	.0756	1.9mm .0748	.034052	.0767	1.95mm .0763	.052073	.0779	5/64 .0781	.073095	.0785	#47 .0785	.095169	.0790	2mm .0787
3-48	020040	.8680	2.2mm .0866	.040059	.0882	#43 .089	.059084	.0895	#43 .089	.084110	.0902	2.3mm .0906	.110141	.0909	2.3mm .0906
4-40	.022045	.0974	#40	.045067	.0990	#39	.067095	.1006	#39 .0995	.095126	.1014	#38 .1015	.126157	.1023	2.6mm
5-40	.025051	.1104	2.8mm	.051075	.1120	#33	.075106	.1136	#33 .113	.106141	.1144	2.9mm .1142	.141175	.1153	2.9mm .1142
6-32	.028066	.1197	#31 .120	.066083	.1218	3.1mm .122	.083117	.1238	1/8 .125	.117152	.1248	1/8 .125	.152193	.1258	3.2mm .126
8-32	.033066	.1457	3.7mm .1457	.066098	.1478	3.75mm .1476	.098141	.1498	3.8mm .1496	.141-1.80	.1508	3.8mm .1491	.180230	.1518	#24 .152
10-24	.038079	.1656	#19 .166	.079114	.1683	#18 .1695	.114162	.1710	11/64 .1719	.162209	.1724	11/64 .1719	.209266	.1738	4.4mm .1732
10-32	.038079	.1717	11/64 .1719	.079114	.1738	#17 .173	.114162	.1758	#16 .177	.162209	.1768	#16 .177	.209266	.1778	4.5mm .1772
12-24	.043086	.1916	#11 .191	.086130	.1943	#9 .196	.130184	.1970	5mm .1969	.184238	.1984	#8 .199	.238302	.1998	5.1mm .2008
1/4-20	.050100	.2208	#2 .221	.100150	.2240	5.7mm .2244	.150213	.2273	#1 .228	.213275	.2289	5.8mm .2283	.275350	.2309	5.8mm .2283
5/16-18	.062126	.2800	7.1mm .2795	.126188	.2836	7.2mm .2835	.188266	.2872	7.3mm .2874	.266345	.2890	L .29	.345438	.2908	7.4mm .2913
3/8-16	.075150	.3384	8.6mm .3386	.150225	.3425	8.7mm .3425	.225319	.3466	8.8mm .3465	.319413	.3486	Size .348	.413525	.3506	8.9mm .3504
7/16-14	.087174	.3957	X .397	.174262	.4004	X .397	.262371	.4050	Y .404	.371481	.4073	13/32 .4063	.481612	.4096	13/32 .4063
1/2-13	.100200	.4550	29/64 .4531	.200300	.4600	29/64 .4531	.300425	.4650	15/32 .4688	.425550	.4675	15/32 .4688	.550700	.4700	15/32 .468

APPLICATION DUTY CLASS – A general term used here to group material thickness in terms of screw diameters. For example, the average material thickness listed under "medium-heavy" equals 75% of the screw diameter.

Recommended Extruded Pilot Hole Sizes in Light-Gage Steel



Hole Size Dia. (D) per Material Thickness - Metric Sizes (mm)

	Material Thickness										
Screw Size	0.50 - 0.69	0.70 - 0.99	1.00 - 1.49	1.50 - 2.49	2.50 - 3.00						
M2.5 x 0.45	2.22	2.23	2.24	_	_						
M3 x 0.5	2.70	2.71	2.72	_	_						
M4 x 0.7	3.57	3.59	3.61	3.64	_						
M5 x 0.8	_	4.53	4.56	4.59	_						
M6 x 1.0	_	5.42	5.45	5.48	5.51						
M8 x 1.25	_	_	7.27	7.31	7.35						

Extruded Hole Thicknesses - Metric Sizes (mm)

Metric	Approximate Material Thickness "T"										
Hole Dia.	0.6	- 1.0	1.0 -	- 1.2	1.2	- 2.0	2.0	- 2.5	2.5 - 3.0		
D	Н	R	Н	R	Н	R	Н	R	Н	R	
2.00 - 2.55	1.00	0.13	1.00	0.13	1.00	0.15	1.10	0.25	_	_	
2.56 - 3.20	1.20	0.13	1.20	0.13	1.20	0.15	1.30	0.25	1.35	0.25	
3.21 - 3.80	1.35	0.13	1.35	0.13	1.35	0.15	1.50	0.25	1.60	0.25	
3.81 - 4.60	_	_	1.50	0.13	1.55	0.15	1.80	0.25	1.90	0.25	
4.61 - 5.60	_	_	1.80	0.13	1.80	0.15	2.30	0.25	2.40	0.25	
5.61 - 6.60	_	_	_	_	1.90	0.15	2.55	0.25	2.65	0.25	
6.61 - 7.60					2.10	0.15	2.95	0.25	3.20	0.25	

Hole Size Diameter (D) per Material Thickness – Inch Sizes (in.)

	Material Thickness										
Screw Size	.020 – .029	.030 – .039	.040 – .059	.060 – .099	.100 – .130						
4-40	.097	.097	.098	_	_						
6-32	.119	.120	.121	.122	_						
8-32	.145	.146	.147	.148	_						
10-24	.164	.166	.168	.170	.170						
10-32	.171	.172	.173	.174	.174						
1/4-20	_	.221	.223	.225	.225						
5/16-18	_	_	.282	.285	.285						

Extruded Hole Thicknesses - Inch Sizes (in.)

		Approximate Material Thickness "T"											
Inch Hole	.020-	.035	.035	050	.050	075	.075	100	.100125				
Dia. (D)	Н	R	Н	R	Н	R	Н	R	Н	R			
.081100	.040	.005	.040	.005	.040	.006	.043	.010	_	_			
.101125	.047	.005	.047	.005	.047	.006	.052	.010	.054	.010			
.126150	.053	.005	.053	.005	.053	.006	.060	.010	.063	.010			
.151180	_	_	.060	.005	.060	.006	.070	.010	.075	.010			
.181220	_	_	.070	.005	.070	.006	.090	.010	.095	.010			
.221260	_	_	_	_	.0075	.006	.100	.010	.105	.010			
.261300	_	_	_	_	.083	.006	.116	.010	.125	.010			

Extruding holes for fasteners in light-gage steel nearly doubles the length of thread engagement over the original material thickness.

TAPTITE 2000® fasteners develop almost twice the failure torque in extruded holes, providing maximum joint integrity.

Example: The chart shows that for a M4 x 0.7 screw in a material thickness of 0.75mm the suggested hole diameter is 3.59mm. The corresponding "H" dimension is the 1.35mm minimum, making the total length of engagement 2.1mm minimum.

Hole Sizes per Percentage of Thread Engagement – Metric Sizes (mm)

Nominal	Percent Thread Engagement													
Screw Size	100	95	90(1)	85(1)	80	75	70	65	60	55	50	45	40	35
M2.5 x 0.45	2.21	2.22	2.24	2.25	2.27	2.28	2.29	2.31	2.32	2.34	2.35	2.37	2.38	2.40
M3 x 0.5	2.67	2.69	2.71	2.72	2.74	2.76	2.77	2.79	2.80	2.82	2.84	2.85	2.87	2.90
M3.5 x 0.6	3.11	3.13	3.15	3.17	3.19	3.21	3.23	3.25	3.27	3.29	3.30	3.32	3.34	3.36
M4 x 0.7	3.54	3.57	3.59	3.61	3.64	3.66	3.68	3.70	3.73	3.75	3.77	3.79	3.80	3.84
M4.5 x 0.75	4.01	4.04	4.06	4.09	4.11	4.13	4.16	4.18	4.21	4.23	4.26	4.28	4.30	4.33
M5 x 0.8	4.48	4.51	4.53	4.56	4.58	4.61	4.64	4.66	4.69	4.71	4.74	4.77	4.79	4.82
M6 x 1.0	5.35	5.38	5.42	5.45	5.48	5.51	5.54	5.58	5.61	5.64	5.67	5.71	5.74	5.77
M7 x 1.0	6.35	6.38	6.42	6.45	6.48	6.51	6.54	6.58	6.61	6.64	6.67	6.71	6.74	6.77
M8 x 1.25	7.19	7.23	7.27	7.31	7.35	7.39	7.43	7.47	7.51	7.55	7.59	7.63	7.67	7.72
M10 x 1.5	9.03	9.07	9.12	9.17	9.22	9.27	9.32	9.37	9.41	9.46	9.51	9.56	9.61	9.66
M12 x 1.75	10.86	10.92	10.98	11.30	11.09	11.15	11.20	11.26	11.31	11.37	11.43	11.49	11.55	11.60

Hole Sizes per Percentage of Thread Engagement – Inch Sizes (in.)

Nominal	Percent Thread Engagement													
Screw Size	100	95	90(1)	85(1)	80	75	70	65	60	55	50	45	40	35
2-56	.0744	.0750	.0756	.0761	.0767	.0773	.0779	.0785	.0790	.0796	.0802	.0808	.0814	.0819
3-48	.0855	.0861	.0868	.0875	.0882	.0888	.0895	.0902	.0909	.0916	.0922	.0929	.0936	.0943
4-40	.0958	.0966	.0974	.0982	.0990	.0998	.1006	.1014	.1023	.1031	.1039	.1047	.1055	.1063
5-40	.1088	.1096	.1104	.1112	.1120	.1128	.1136	.1144	.1153	.1161	.1169	.1177	.1185	.1193
6-32	.1177	.1187	.1197	.1207	.1218	.1228	.1238	.1248	.1258	.1268	.1278	.1289	.1299	.1309
8-32	.1437	.1447	.1457	.1467	.1478	.1488	.1498	.1508	.1518	.1528	.1538	.1549	.1559	.1569
10-24	.1629	.1643	.1656	.1670	.1683	.1697	.1710	.1724	.1738	.1751	.1765	.1778	.1792	.1805
10-32	.1697	.1707	.1717	.1727	.1738	.1748	.1758	.1768	.1778	.1788	.1798	.1809	.1819	.1829
12-24	.1889	.1903	.1916	.1930	.1943	.1957	.1970	.1984	.1998	.2011	.2025	.2038	.2052	.2065
1/4-20	.2175	.2191	.2208	.2224	.2240	.2256	.2273	.2289	.2305	.2321	.2338	.2354	.2370	.2386
5/16-18	.2764	.2782	.2800	.2818	.2836	.2854	.2872	.2890	.2908	.2926	.2944	.2963	.2981	.2999
3/8-16	.3344	.3364	.3384	.3405	.3425	.3445	.3466	.3486	.3506	.3527	.3547	.3567	.3588	.3608
7/16-14	.3911	.3934	.3957	.3980	.4004	.4027	.4050	.4073	.4096	.4120	.4143	.4166	.4189	.4213
1/2-13	.4500	.4525	.4550	.4575	.4600	.4625	.4650	.4675	.4700	.4725	.4750	.4775	.4800	.4825

NOTE: Nominal screw diameters are used when calculating hole sizes and are based on the U.S. basic thread depth of 0.6495 times the pitch. Hole data accuracy decreases for engagements less than 70%. This is because the above data is based on a linear relation between hole size and percentage of thread engagement.

Hole Size = $D - (0.6495 \times P \times \%)$. In this equation, D is equal to the nominal screw diameter.

(1) Pilot holes listed under the 90% and 85% thread engagement columns are recommended for single punch extruded holes.

Typical tolerance for the pilot hole range is -10% to +5% from the nominal percent thread engagement.

Example: for a M12 TAPTITE 2000® fastener using a single punch extruded hole, the nominal hole size is 85% thread engagement or 11.30mm, bounded by a tolerance window of 11.15mm (75% thread engagement) to 10.98mm (90% thread engagement.

PROPER DESIGN

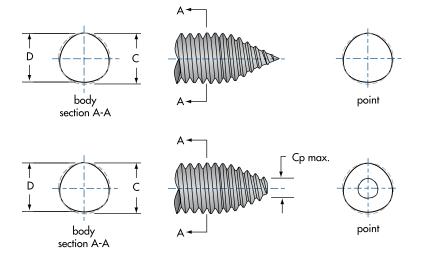
Due to the variability of materials, please contact an STANLEY Engineered Fastening applications engineer for proper hole size recommendations to ensure proper fastener performance.

Special Point Designs

TAPTITE 2000® CA™ Point Fasteners

TAPTITE 2000® CA fasteners have a gimlet point for use where clearance holes and pilot holes do not align. The CA point is also good for rapid hole finding, floating nut members or difficult access applications.



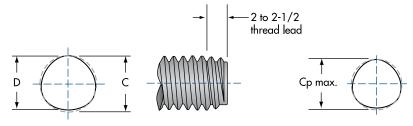


The CA point can be supplied with a sharp point or a slightly truncated blunt point, which is desirable for situations when the sharp point could be a potential hazard to wires, components or assembly line and service personnel.

Taptite 2000® "SP"™ (Short Point) Fasteners

TAPTITE 2000® "SP" fasteners have a shorter point than standard TAPTITE 2000® fasteners to maximize full thread engagement in blind holes, particularly in nonferrous materials such as aluminum.

The short point of the TAPTITE 2000® "SP" fastener maximizes the amount of full thread engagement in blind holes. Increasing the full thread engagement is often critical in shallow depth holes. In many cases the failure mode can be shifted from internal thread stripping to fastener fracture, which is typically desired in castings. In deeper holes, the shorter "SP" point may allow use of a shorter fastener, saving weight and cost.







TAPTITE 2000® "SP"™ Fasteners

Recommended Pilot Hole Sizes for Aluminum or Zinc Alloy Die Castings for TAPTITE 2000® "SP™ Fasteners

The minimum length of thread engagement should be equal to twice the diameter of the screw (to approach utilizing available screw strength). To ensure optimum performance, the hole diameter should provide for 65% to 75% thread engagement.

Metric Sizes (mm)

		Hole Diam	eter as Cast		-	L Length	н	Distance to Edge for no measurable
	Top A		Bottom B		Hole Dia. as	of Thread	Boss Dia.	Distortion
Screw Size	max.	min.	max.	min.	Drilled	Engagement	min.	min.
M2 x 0.40	1.91	1.83	1.81	1.73	1.82	4.00	3.32	1.0
M2.5 x 0.45	2.39	2.31	2.28	2.20	2.29	5.00	4.15	1.2
M3 x 0.5	2.90	2.82	2.76	2.68	2.77	6.00	4.98	1.3
M3.5 x 0.6	3.31	3.23	3.21	3.13	3.23	7.00	5.81	1.6
M4 x 0.7	3.82	3.74	3.64	3.56	3.68	8.00	6.64	1.8
M5 x 0.8	4.80	4.72	4.58	4.50	4.64	10.00	8.30	2.1
M6 x 1.0	5.74	5.66	5.48	5.40	5.54	12.00	9.96	2.6
M7 x 1.0	6.78	6.70	6.48	6.40	6.54	14.00	11.62	2.6
M8 x 1.25	7.69	7.61	7.35	7.27	7.43	16.00	13.28	3.3
M10 x 1.5	9.64	9.56	9.22	9.14	9.32	20.00	16.60	3.9
M12 x 1.75	11.59	11.51	11.09	11.01	11.20	24.00	19.92	4.6

Inch Sizes (in.)

		Hole Diam	eter as Cast		F	L Length	н	Distance to Edge for no measurable
	Top A		Bottom B		Hole Dia. as	of Thread	п Boss Dia.	Distortion
Screw Size	max.	min.	max.	min.	Drilled	Engagement	min.	min.
2-56	.081	.078	.077	.074	.0779	.172	.197	.046
3-48	.093	.090	.088	.085	.0895	.198	.208	.054
4-40	.105	.102	.099	.096	.1006	.224	.220	.065
5-40	.118	.115	.112	.109	.1136	.250	.232	.065
6-32	.128	.125	.122	.119	.1238	.276	.242	.081
8-32	.155	.152	.148	.145	.1498	.328	.272	.081
10-24	.177	.174	.168	.165	.1710	.380	.315	.108
10-32	.182	.179	.174	.171	.1758	.380	.315	.081
12-24	.203	.200	.194	.191	.1970	.432	.359	.108
1/4-20	.235	.232	.224	.221	.2273	.500	.415	.130
5/16-18	.297	.294	.284	.281	.2872	.625	.519	.144
3/8-16	.359	.356	.343	.340	.3466	.750	.623	.162
7/16-14	.419	.416	.400	.397	.4050	.875	.726	.186
1/2-13	.481	.478	.460	.457	.4650	1.000	.830	.200

Lower In-Place Costs

Through the elimination of tapping operations and their optimal performance, TAPTITE 2000® fasteners can lower your installed costs. Have you considered these design and assembly issues that TAPTITE 2000® fasteners can eliminate or improve?

Savings Through Elimination of Ta Operations	pping	Savings Through Increased Performance of TAPTITE 2000® Fasteners				
Advantage	Est. Cost Savings	Advantage	Est. Cost Savings			
☐ Direct labor for tapping operations		\square Uniform driving torque, low end				
☐ Indirect labor for tapping operations		requirements, and high drive-to-f	ail			
☐ Taps		☐ Reduced assembly problems from				
☐ Jogs and fixtures		stripped threads and unseated				
☐ Tapping lubricants		fasteners				
☐ Gauges		 Lowered tool costs from extended driver, bit and socket life 	<u></u>			
☐ Set-up time of tapping equipment		. ☐ Reduced operator fatigue				
☐ Downtime on automated equipment due to tapping station malfunction		☐ Eliminated use of waxes or lubrica	ants			
☐ Downtime to replace broken taps		☐ Superior vibration resistance and excellent torque-tension relations	ship			
☐ Low machine efficiency from loading, galling and binding of taps		☐ Smaller diameter or fewer screws be used	•			
☐ Cleaning of oil and chips		☐ Can replace threaded inserts				
☐ Inspection for class of fit		☐ Can be used as an adjusting or				
\square Loss or repair of tapped assemblies due		calibrating screw				
to under- or over-sized tapped threads Loss or repair of tapped assemblies		☐ Can be made captive without cos secondary operations	tly 			
due to tap breakage ☐ Moving components to and from		☐ Elimination of lock nuts, washers other secondary locking devices	and 			
tapping department		☐ Reduced need for weld and clinch nuts by utilizing extruded holes	1			
☐ Use of drilling and tapping stations for other needed operations		☐ Greater use of die casting and oth	ner			
☐ Improved cycle time on multi- operation equipment		soft materials ☐ Improved quality and joint integr	ity			
☐ Elimination of cross-threading into pre-tapped holes		☐ Short point available				
☐ Elimination of tapped holes clogged with paint or other foreign materials		☐ Shorter holes can be used where length is restricted	hole 			
☐ Reduction of drilling operations – holes can be cored or punched		Total savings from improved performance of TAPTITE 2000® faste	ners			
during part blanking		Overall Savings				
☐ Allows largest diameter punch in each screw size for less breakage and longer life		Total savings from elimination of tapping operations				
☐ Can thread directly into less expensive untapped tubular rivets and bushings		Total savings from improved performance of TAPTITE 2000®				
Total savings from elimination of tapping operations		fasteners				
		Total savings				
		No of gained advantages				

STANLEYEngineered Fastening



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